



**The Most Durable
Automated Transmission**

Overview

Nissan Experience

Torotrak Experience

DFTV Durability

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Because the DFTV based IVT can be so compact and light weight when durability is important the transmission size can be increased so that durability is increased.

The durability of the various transmission components in any form of transmission are very dependent on the quality of the fluid used in the transmission and the ability to keep the fluid clean and the ability to keep the fluids cool.

The Traction fluids used in CVT's are some of the best available and the inherent efficiency ensures the fluid is not thermally stressed. The filter system and oil circulation in a CVT ensures the fluid is kept exceptionally clean. Typical CVT's require oil changes at around more than 200,000 kms.

The friction clutches in a non IVT transmission are subject to high contact speeds as the transmission steps through the required ratios. High speeds create wear within the clutch plates and are a source of pollution of the lubricants.

In an IVT transmission the contact speeds of the clutch plates is always low as there are no steps and no launching process requiring plates to slip over each other at high speed.

The durability of the CVT components is critical to the overall durability of a transmission using a CVT.

Nissan Experience

The Nissan NSK transmissions that were introduced into the first cars in 2000 have proven their durability performance in real automobile applications. Ultimate Transmissions research has revealed very few transmission failures in under 200,000 kms. The failures that did occur were almost always associated with the use of the wrong traction fluid. Of the rest, it was common for the thrust bearings not the discs and rollers to fail.

The design life of an NSK CVT was 400,000 kms. The calculations made to ensure this design life were based on conventional bearing life calculations taking into account the fact that exceptionally pure steel as being used, and typical passenger vehicle duty cycles.

Torotrak Experience

Torotrak state* that they base the CVT life on 380 million cycles at 2.1 GPa for calculating design life. This equates to a maximum contact stress of 3.15GPa. When a Torotrak variator is operating in low gear at maximum input torque the maximum design stress under the (inner) contact point is 4.2GPa.

Nissan/NSK limit their maximum stress to 3.95 GPa or a Mean stress of 2.63GPa.

A more recent Torotrak paper expresses the following view of roller and disc durability.

The most recent development is the results from extended testing at lower contact stresses in the area of ~ 2.0GPa mean Hertzian contact stress. Here, a number of test results have been produced with fatigue life extending well beyond the 12th power prediction. This leads to the potential for extended (if not potentially infinite) life of the variator below a contact stress threshold – see figure 14.

The impact is extended durability and, in conjunction with powersplit transmission architectures, further downsizing of the variator and hence transmission.

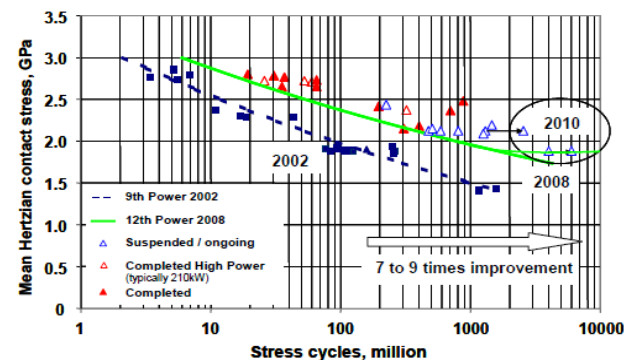


Fig. 14. S-N curves from fatigue tests

* Extract from paper presented by Torotrak (Development) Ltd. Jtekt Corp & Shell Global Solutions to the JSME International Conference on Motion and Power Transmissions May 13-14, 2009 Matsushima Resort Japan entitled "FULLTOROIDAL TRACTION DRIVE HIGH TEMPERATURE DURABILITY" By Dr Adrian Lee, Dr Andrew Hillsden, Yoshihiro Ono and Dr Stephen Evans.

Figure 9. Extract from Paper by Torotrak (Development) Ltd. "Full Toroidal Traction Drives for Front Wheel Drive Applications" by Chris Brockbank & Dave Burt 2006

It can be seen from this diagram that most of the duty is associated with very low wheel torque and low speed. In this condition Ultimate Transmission estimates the Mean contact stresses as being less than 1GPa. In this condition a life exceeding 3,800 million cycles is expected. At 40 kms/hr. this equates to over 1,500,000 kms.

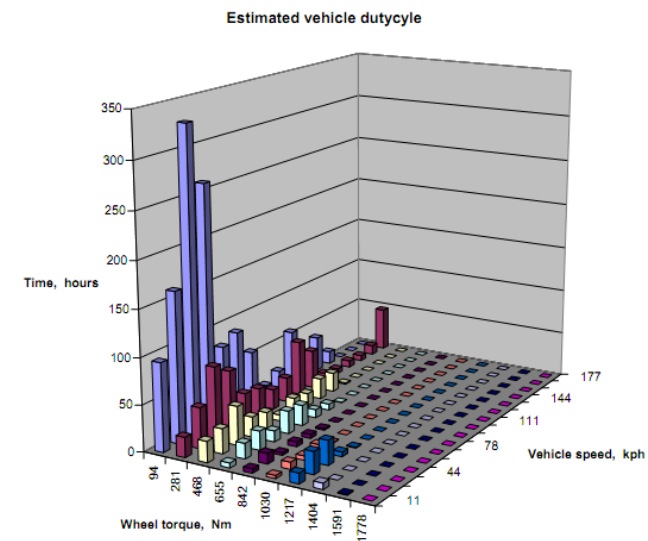


Figure 9 – Small front Wheel Drive Car Duty Cycle 160,000 kms.

Ultimate Transmission's designs for the DFTV variator limit the maximum Hertzian stresses to 3.95GPa (Mean 2,63GPa) which is more in line with the NSK limits.

CALCULATION OF SPIN LOSSES MAXIMUM INPUT TORQUE AND MECHANICAL VOLUME FOR TORQUE CONTROLLED VARIATOR DOUBLE ROLLER

The following model can be used to calculate 1) the maximum spin losses, 2) The maximum ratio, 3) Mechanical Volume 4) Maximum input torque. 5) Estimated weight

Primary inputs are shown as			Secondary inputs from Hertzian contact calculator are shown as			Calculated Outputs								
INPUTS														
Radius of centre of Toroid			RR	67.07	mm.	INPUT RPM		6000	RPM					
Transverse radius of Toroid			R	50.92	mm.	Width of contact point on input	W1	9.85	mm.					
Wheel inclination			a	21.70	deg	Width of contact point on output	W2	9.36	mm.					
Radius of roller			r1	24.80	mm.	OUTPUT RPM		2,561	RPM					
Primary rotation angle			A	31.95	deg	ROLLER RPM		9,707	RPM					
Radius of crown of roller				38.70	mm.	TOTAL RATIO CHANGE		5.49						
Conformity of wheel to cavity				76.00	%	Area of inner contact point		19.21	sq.mm					
Length of input contact point			L1	2.48	mm.	Maximum Hertzian Stress		3.85	Gpa					
Number of wheels				4	rollers	Maximum stress on inner point from Hertzian table		49,514	N					
Number of cavities				1	cavity	Traction coefficient		0.055						
Length of output contact point			L2	3.2016		maximum reaction force on roller bearings		5,447	N					
	R1	R2	R3	r1	r2	r3	Sin A	Cos A	Sin a	Cos a	Size factor	Total clamping force	198,058	N
	40.1211469	44.3014173	35.9408765	24.7986	26.6202	22.9770	0.529179	0.84851021	0.36974676	0.92913257	1.38	stress under output contact	3.17	
	X1	X2	X3	r1	x2	x3	Sin A	Cos A	Sin a	Cos a		Overall radius of transmission without actuator	107.07	
	94.0148531	97.9843536	90.0453526	24.7986	26.5283	23.0689	0.529179	0.84851021	0.36974676	0.92913257		Overall length of transmission without hydraulics	165.60	
												Overall length between outside of discs	122.21	
							Speed dif	% loss	Spin for this position			MAXIMUM INPUT TORQUE		spin
Speed at	R1	1,511,765		r1	1,511,765	mm/minute	0					437.05	Nm	
Speed at	R2	1,669,277		r2	1,622,812		46,465	3.07%	5.83%			ESTIMATED MASS		
Speed at	R3	1,354,252		r3	1,400,717		-46,465	3.07%				ESTIMATED VOLUME		
												35.05	kg	7.15
Speed at	X1	1,511,765		r1	1,511,765		0		% of SFTV			4.40	Discs and rollers only	litres
Speed at	X2	1,575,595		x2	1,617,213		-41,619	2.75%	63.82%			KW		torque density
Speed at	X3	1,447,935		x3	1,406,316		41,619	2.75%				274.58		99.340317
CALCULATION OF APPARENT RADIUS OF CONTACTING POINTS														
Angle of inclination of circle of contact of toroidal disc				A	31.95	deg	Spin loss in "L" position		5.83%			ENERGY LOST TO SPIN		
Angle of inclination of circle of contact of roller				a	21.70	deg	Spin loss in "LM" position		3.58%			2.23% factor	0.675	
Radius of input circle of contact on toroidal disc					75.82	mm.	Spin loss in "C" position		0.00%			ENERGY LOST TO BEARING		
Radius of output circle of contact of toroidal disc					-177.66	mm.	Spin loss in "HM" position		3.19%			0.00%		
Radius of contact of (toroidal) roller surface					26.69	mm.	Spin loss in "H" position		7.17%			TOTAL LOSS		
							Average spin loss		3.30%			2.23%		
							VOLUME OF OPERATIONAL CAVITIES		4.45	litres				

The extended life available from properly designed rolling contacts combined with the limited use of gears and clutches and associated lower parts count all go to verifying the superior life expectations of a DFTV - IVT.

The DFTV - IVT can remain smaller and more efficient than other transmission designs while maintaining greater life expectancy.

Extract from Ultimate Transmissions Excel based Traction performance calculator.

